rldesign.PLeLa Proportional-lead-lag controller design

Proportional-lead-lag controller design is much like PID controller design, but the result<u>ing</u> controller does not require active compensation. With our techniques of cascade compensation for lead and lag compensators, one can simply apply both lead and lag compensation in the usual manner. The order of application can be somewhat important because lead compensation can impact steady-state error. A way to proceed is as follows.

- 1. Design a P controller and evaluate its transient response performance.
- 2. Apply lead compensation to improve the transient response. Simulate to verify the transient response performance.
- 3. Apply lag compensation to improve the steady-state error performance.
- Check all performance criteria and adjust gains and zero locations, as-needed.

A design example

Let a system have plant transfer function

$$\frac{200}{s^3 + 29s^2 + 170s - 200}.$$
 (1)

Design a P-lead-lag controller such that the closed-loop overshoot is less than 20%, settling time is less than 0.7 seconds, and the steady-state error is less than 3%.

$\texttt{Determining}\; \psi$

We use Matlab for the design.¹⁰ First, we must determine what the specified transient response criteria imply for the locations of our

10. See ricopic.one/control/source/plaglead_controller_design_example.m for the source.

closed-loop poles. Let one of these desired pole locations be called ψ . The transient response performance criteria are as follows.

Ts = .7; % sec ... spec settling time
OS = 20; % percent ... spec overshoot
sse = .03; % fraction of 1

The second-order approximation from Chapter trans tells us that the overshoot requirement implies a specific damping ratio ζ , or, equivalently, $\angle \psi$:

$$\angle \psi = \pi - \arccos \zeta.$$
 (2)

Additionally, the settling time requirement implies a specific $\text{Re}(\psi)$ via

$$T_{\rm S} = -4/\,{\rm Re}(\psi). \tag{3}$$

psi = -5.71 + j 11.2

P control

We design a proportional controller that gets us as close as possible to ψ . The root locus is shown in Figure PLeLa.1.



Figure PLeLa.1: root locus without compensation.

```
G = tf([200],[1,29,170,-200]);
figure
rlocus(G)
```

Although we cannot get close to ψ on the root locus, we can at least meet our %OS specification by choosing a gain of about

$$K_1 = 5.$$
 (4)

Let's construct the compensator and corresponding closed-loop transfer function G_P for gain control.

K_1 = 5; G_P = feedback(K_1*G,1);

Lead compensation

Now, we use cascade lead compensation with compensator

$$K_{2} \frac{s - \mathbf{z}_{ld}}{s - p_{ld}}.$$
 (5)

For now, we set $K_{\rm Q}$ = 1. Let's also set, arbitrarily, $\rho_L d$ = -30. From Eq. 5b, we compute

the compensator zero

 $\theta_{c} = \pi - \angle G(\psi)$ and $\mathbf{z}_{c} = \operatorname{Re}(\psi) - |\operatorname{Im}(\psi)| / \tan(\theta_{c} + \angle(\psi - \rho_{c}))$.

```
p_ld = -30;
theta_ld = pi - angle(evalfr(G,psi));
theta_p_ld = angle(psi-p_ld);
z_ld = real(psi) - abs(imag(psi))/tan(theta_ld + theta_p_ld);
disp(sprintf('theta_ld = %0.3g deg',rad2deg(theta_c)))
disp(sprintf(...
    'pole phase contribution = %0.3g deg',...
    rad2deg(theta_p_c)...
))
disp(sprintf('z_ld = %0.3g',z_ld))
```

```
theta_ld = 48 deg
pole phase contribution = 24.7 deg
z_1d = -9.19
```

By construction, ψ is on the root locus, so we can find K_{2} directly from Eq. 2.

```
C_sans = zpk(z_ld,p_ld,1); % without gain
K_2 = 1/abs(evalfr(K_1*C_sans*G,psi));
C_ld = K_1*K_2*C_sans;
disp(sprintf('K_2 = %0.3g',K_2))
```

 $K_2 = 6.45$

Let's compute the closed-loop controller $C_{\text{lead}},$ and the closed-loop transfer function $G_{\text{lead}}.$

G_Plead = feedback(C_ld*G,1);

Lag compensation

Now, we use cascade lag compensation with compensator

$$K_3 \frac{s - \mathbf{z}_{lg}}{s - p_{lg}}.$$
 (6)

For now, we set $K_3 = 1$.

The steady-state error for the lead compensated system is given by the following.

```
Kp_ld = evalfr(C_ld*G,0);
ess_ld = 1/(1+Kp_ld);
disp(sprintf('steady-state error = %0.3g',ess_ld))
```

```
steady-state error = -0.113
```

The negative value implies the output is larger than the input. Reducing this to the given requirement implies an approximate ratio of compensator zero to pole α , as follows.

alpha = abs(ess_ld)/sse

alpha =

3.7533

If we begin, somewhat arbitrarily, with p_{lg} and $z_{lg} = \alpha p_{lg}$. Let's construct the compensator and closed-loop transfer function G_{PLL}.

```
p_lg = -.1;
z_lg = alpha*p_lg;
C_sans = zpk(z_lg,p_lg,1);
G_PLL = feedback(C_sans*C_ld*G,1);
```

Simulate

Our placement of the ψ depended on the second-order approximation's accuracy. In any case, we simulate the step response to test the efficacy of the P-lead and P-lead-lag controller designs and compare them with the P controller.

t_a = linspace(0,2.5,200); % s ... sim time
y_P = step(G_P,t_a); % P controlled step response
y_Plead = step(G_Plead,t_a); % P-lead step resp.
y_PLL = step(G_PLL,t_a); % P-lead-lag step resp.

```
figure
plot(t_a,y_P);hold on;
plot(t_a,y_Plead);
plot(t_a,y_PLL);
xlabel('time (s)');
ylabel('step response');
grid on
legend(...
    'P control','P-lead','P-lead-lag',...
    'location','southeast'...
);
```

The responses, shown in Figure PLeLa. 2, suggest



Figure PLeLa.2: step responses for proportional, proportional-lead, and proportional-lead-lag controllers.

the lead and lead-lag compensated controllers nearly meet the transient requirements. Let's use stepinfo to compute more accurate transient response characteristics for the different controllers.

```
disp('P control')
si_P = stepinfo(y_P,t_a);
disp(sprintf('settling time: %0.3g',si_P.SettlingTime))
disp(sprintf('percent overshoot: %0.3g\n',si_P.Overshoot))
si_Plead = stepinfo(y_Plead,t_a);
disp('P-lead control')
disp(sprintf(...
  'settling time: %0.3g',si_Plead.SettlingTime ....
))
disp(sprintf(...
  'percent overshoot: %0.3g\n',si_Plead.Overshoot...
))
si_PLL = stepinfo(y_PLL,t_a);
disp('P-lead-lag control')
disp(sprintf(...
  'settling time: %0.3g',si_PLL.SettlingTime ...
))
disp(sprintf(...
  'percent overshoot: %0.3g\n',si_PLL.Overshoot...
))
```

P control settling time: 1.41 percent overshoot: 16

P-lead control settling time: 0.689 percent overshoot: 17.2

P-lead-lag control settling time: 1.57 percent overshoot: 25.1

The stepinfo results are not very precise for the P-lead-lag controller due to the slow steady-state compensation, which isn't completely finished by the end of the simulation. Adjusting compensator zeros and poles may improve things, but a trade-off emerges between overshoot and steady-state compensation: speeding up the latter increases the overshoot rather sharply.

The steady-state requirement can be checked analytically.

```
Kp_PLL = evalfr(C_sans*C_ld*G,0);
ess_PLL = 1/(1+Kp_PLL);
disp(sprintf('steady-state error = %0.3g',ess_PLL))
```

steady-state error = -0.0277

This is less than 3%, per the requirement; however, the compensation does take a relatively long time to approach this small error.