08.1 thermoflu.flu Fluid system elements

1 Detailed *distributed* models of fluids, such as the Navier-Stokes equations, are necessary for understanding many aspects of fluid systems and for guiding their design (e.g. a pump or an underwater vehicle). However, a great many fluid systems are networks of pipes, tanks, pumps, valves, orifices, and elevation changes—and at this *system-level*, a different approach is required.

2 As with electrical and mechanical systems, we can describe fluid systems as consisting of discrete lumped-parameter elements. The dynamic models that can be developed from considering these elements are often precisely the right granularity for system-level design.

3 We now introduce a few lumped-parameter elements for modeling fluid systems. Let a **volumetric flowrate** Q and **pressure drop** P be input to a port in a fluid element. Since, for fluid systems, the power into the element is

$$\mathcal{P}(t) = Q(t)P(t) \tag{1}$$

we call Q and P the **power-flow variables** for fluid systems. A fluid element has two distinct locations between which its pressure drop is defined. We call a reference pressure **ground**.

4 Work done on the system over the time interval [0, T] is defined as

$$W \equiv \int_0^T \mathcal{P}(\tau) d\tau.$$
 (2)

Therefore, the work done on a fluid system is

$$W = \int_0^T Q(\tau) P(\tau) d\tau.$$
(3)

5 The **pressure momentum** Γ is

$$\Gamma(t) = \int_0^t P(\tau) d\tau + \Gamma(0).$$
(4)

Similarly, the **volume** is

$$V(t) = \int_0^t Q(\tau) d\tau + V(0).$$
(5)

6 We now consider two elements that can store energy, called **energy storage elements**; an element that can dissipate energy to a system's environment, called an **energy dissipative element**; and two elements that can supply power from outside a system, called **source elements**.

Fluid inertances

7 When fluid flows through a pipe, it has a momentum associated with it. The more mass (fluid density by its volume) moving in one direction and the faster it moves, the more momentum. This is stored kinetic energy. The discrete element we now introduce models this aspect of fluid systems.

8 A **fluid inertance** is defined as an element for which the pressure momentum Γ across it is a monotonic function of the volumetric flowrate Q through it. A **linear inertance** is such that

$$\Gamma(\mathbf{t}) = \mathrm{IQ}(\mathbf{t}),\tag{6}$$

where I is called the **inertance** and is typically a function of pipe geometry and fluid properties. This is called the element's **constitutive equation** because it constitutes what it means to be an inertance.

9 Although there are nonlinear inertances, we can often use a linear model for analysis in some operating regime. The **elemental equation** for a linear inertance can be found by time-differentiating Equation 6 to obtain

We call this the elemental equation because it relates the element's power-flow variables Q and P.

10 An inertance stores energy as kinetic energy, making it an *energy storage element*. The amount of energy it stores depends on the volumetric flowrate

it contains. For a linear inertance,

$$\mathcal{E}(\mathbf{t}) = \frac{1}{2} \mathrm{IQ}(\mathbf{t})^2. \tag{7}$$



Figure flu.1: a section of pipe for deriving its inertance.

11 The inertance I for a uniform pipe can be derived, as follows, with reference to the sectioned pipe of Fig. flu.1. For an incompressible fluid flowing through a pipe of uniform area A and length L, with uniform velocity profile (a convenient fiction), an element of fluid obey's Newton's second law, from which several interesting equalities can be derived:

$$F = m\frac{dv}{dt} \Rightarrow$$

$$\frac{F}{A} = \frac{m}{A}\frac{dv}{dt} \Rightarrow$$

$$P = \frac{\rho A L}{A}\frac{dv}{dt}$$

$$= \rho L\frac{d}{dt}\left(\frac{Q}{A}\right)$$

$$= \frac{\rho L}{A}\frac{dQ}{dt} \Rightarrow$$

$$\frac{dQ}{dt} = \underbrace{\frac{A}{\rho L}}_{1/I}P.$$

12 From this last equality, it is clear that, for a uniform pipe and the assumptions, above,

$$I = \frac{\rho L}{A}.$$
 (8)

Clearly, *long*, *thin* pipes will have more inertance. In fact, we often ignore inertance in modeling a pipe, unless it is relatively long and thin.

Fluid capacitors

13 When fluid is stored in tanks or in pressure vessels, it stores potential energy via its pressure drop P. For instance, a tank with a column of fluid will have a pressure drop associated with the height of the column. This is analogous to how an electronic capacitor stores its energy via its voltage. For this reason, we call such fluid elements **fluid capacitors**.

14 A linear fluid capacitor with capacitance C, pressure drop P, and volume V has the constitutive equation

$$V = CP. (9)$$

Once again, time-differentiating the constitutive equation gives us the elemental equation:

15 Fluid capacitors can store energy (making them *energy storage elements*) in fluid potential energy, which, for a linear capacitor is

$$\mathcal{E}(\mathbf{t}) = \frac{1}{2} \mathbf{C} \mathbf{P}^2. \tag{10}$$

Fluid resistors

Fluid resistors are defined as elements for which the volumetric flowrate Q through the element is a monotonic function of the pressure drop P across it. **Linear fluid resistors** have constitutive equation (and, it turns out, elemental equation)

$$Q = \frac{1}{R}P$$
 (11)

where R is called the **fluid resistance**.

16 Fluid resistors dissipate energy from the system (to heat), making them *energy dissipative elements*.

FLUID SYSTEM ELEMENTS

Flowrate and pressure drop sources

17 Fluid sources include pumps, runoff, etc.

18 An **ideal volumetric flowrate source** is an element that provides arbitrary energy to a system via an independent (of the system) volumetric flowrate. The corresponding pressure drop across the element depends on the system.

19 An **ideal pressure drop source** is an element that provides arbitrary energy to a system via an independent (of the system) pressure drop. The corresponding volumetric flowrate through the element depends on the system.

20 Real sources, usually pumps, cannot be ideal sources, but in some instances can approximate them. More typical is to include a fluid resistor in tandem with an ideal source, as we did with electrical resistors for real electrical sources.

Generalized element and variable types

21 In keeping with the definitions of Chapter 01 intro, pressure P is an **across-variable** and flowrate Q is a **through-variable**.

22 Consequently, the fluid capacitor is considered an **A-type** energy storage element. Similarly, the fluid inertance is a **T-type** energy storage element. Clearly, a fluid resistor is a **D-type** energy dissipative element.

23 Pressure sources are, then, across-variable sources and volumetric flowrate sources are through-variable sources.

Example 08.1 thermoflu.flu-1

Use the schematic in Fig. flu.2 to draw a linear graph of the system.



Figure flu.2: schematic of a fluid system for Example 08.3 thermoflu.flutrans-1.

255

re: fluid

system graph